

GUIDANCE RELATING TO THE RULES FOR THE CLASSIFICATION OF STEEL SHIPS

(Guidance of Heat Transfer Analysis for Ships Carrying Liquefied Gases in
Bulk/Ship Using Liquefied Gases as Fuels)

-External Opinion Inquiry-

2020.11.



Hull Rule Development Team

- Main Amendments -

- To reflect Request for Establishment/Revision of Classification Technical Rules

Present	Amendment	Reason
<p style="text-align: center;">CHAPTER 1 GENERAL</p> <p style="text-align: center;">Section 1 Application</p> <p>101. Application</p> <p>1. This guidances applies to the assessment procedure of heat transfer analysis on the hull of ships carrying liquefied gases in bulk with membrane cargo tank and the hull of ships using liquefied gases as fuels with membrane fuel tank.</p> <p>2. ~ 3. <Omitted></p> <p style="text-align: center;">Section 2 ~ Section 4 <Omitted></p>	<p style="text-align: center;">CHAPTER 1 GENERAL</p> <p style="text-align: center;">Section 1 Application</p> <p>101. Application</p> <p>1. This guidances applies to the assessment procedure of heat transfer analysis on the hull of ships carrying liquefied gases in bulk and the hull of ships using liquefied gases as fuels.</p> <p>2. ~ 3. <Same as the present guidance></p> <p style="text-align: center;">Section 2 ~ Section 4 <Same as the present guidance></p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p style="text-align: center;">CHAPTER 2 ANALYTICAL HEAT TRANSFER ANALYSIS</p> <p style="text-align: center;">Section 1 Analysis Procedure</p> <p>101. Procedure of analytical heat transfer analysis</p> <p>1. The analytical heat transfer analysis is performed according to the flowchart in Figure 2.1.</p> <p>(1) As shown in Figure 2.2, the work to divide the compartment for analytical two-dimensional heat transfer analysis into sections is performed.</p> <p>(2) Defines the boundary conditions for the compartment. This includes the settings of the length, width and area of all members of the compartment, seawater temperature, air temperature, wind speed and emissivity of the steel.</p> <p>(3) Assume initial temperature of compartment and member.</p> <p>(4) Calculate the overall heat transfer coefficient.</p> <p>(5) Calculate the temperature of compartment and member.</p> <p>(6) If the change in temperature is below the reference value, the calculation is stopped; otherwise, the calculation is performed in step (4).</p> <p>(7) Perform the above operation to the last compartment to be analyzed.</p> <p>(8) Obtain the hull temperature calculation results.</p> <p>2. Analytical heat transfer analysis method is performed through iterative procedures. In order to reduce the number of repetitions, the initial temperature is set to the atmospheric temperature or the seawater temperature according to the surrounding environment to be contacted. <Omitted></p>	<p style="text-align: center;">CHAPTER 2 HEAT TRANSFER ANALYSIS FOR MEMBRANE TYPE</p> <p style="text-align: center;">Section 1 <u>Analytical Heat Transfer Analysis</u></p> <p>101. Analysis Procedure</p> <p><u>1. Procedure of analytical heat transfer analysis</u></p> <p>(1) The analytical heat transfer analysis is performed according to the flowchart in Figure 2.1.</p> <p>(A) As shown in Figure 2.2, the work to divide the compartment for analytical two-dimensional heat transfer analysis into sections is performed.</p> <p>(B) Defines the boundary conditions for the compartment. This includes the settings of the length, width and area of all members of the compartment, seawater temperature, air temperature, wind speed and emissivity of the steel.</p> <p>(C) Assume initial temperature of compartment and member.</p> <p>(D) Calculate the overall heat transfer coefficient.</p> <p>(E) Calculate the temperature of compartment and member.</p> <p>(F) If the change in temperature is below the reference value, the calculation is stopped; otherwise, the calculation is performed in step (D).</p> <p>(G) Perform the above operation to the last compartment to be analyzed.</p> <p>(H) Obtain the hull temperature calculation results.</p> <p>(2) Analytical heat transfer analysis method is performed through iterative procedures. In order to reduce the number of repetitions, the initial temperature is set to the atmospheric temperature or the seawater temperature according to the surrounding environment to be contacted. <Same as the present guidance></p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p style="text-align: center;">Section 2 Modeling</p> <p>201. 1-Dimensional heat transfer analysis model</p> <p>1. The one-dimensional heat transfer analysis model provides the information necessary to understand the analytical heat transfer analysis method and the two-dimensional model is an extension of the one-dimensional model. The one-dimensional heat transfer analysis model is considered as a horizontal and vertical model and an example is shown in Figure 2.3. <Omitted></p> <p>2. The equilibrium equation of the one-dimensional heat transfer analysis is defined as follows. <Omitted></p> <p>3. The heat transfer of the one-dimensional heat transfer model is defined as follows using the overall heat transfer coefficient. <Omitted></p> <p>4. The overall heat transfer coefficient is obtained by a combination of heat transfer coefficient of convection, conduction and radiation, and the above case is defined as follows.</p> <p>(1) Overall heat transfer coefficient for Q_1 <Omitted></p> <p>(2) Overall heat transfer coefficient for Q_2 <Omitted></p> <p>(3) Heat transfer coefficient(h_C) of convection is calculated as follows. <Omitted></p> <p>(4) Heat transfer coefficient(h_R) of radiation is calculated as follows. <Omitted></p> <p>5. The temperature of the compartment and members is obtained in the same way as the method for obtaining the steel temperature in the example of Figure 2.4 .</p> <p>(1) The heat flux through the outer hull is expressed as follows. <Omitted></p> <p>(2) In equilibrium, the steel temperature is calculated as follows. <Omitted></p>	<p>102. Modeling</p> <p>1. 1-Dimensional heat transfer analysis model</p> <p>(1) The one-dimensional heat transfer analysis model provides the information necessary to understand the analytical heat transfer analysis method and the two-dimensional model is an extension of the one-dimensional model. The one-dimensional heat transfer analysis model is considered as a horizontal and vertical model and an example is shown in Figure 2.3. <Same as the present guidance></p> <p>(2) The equilibrium equation of the one-dimensional heat transfer analysis is defined as follows. <Same as the present guidance></p> <p>(3) The heat transfer of the one-dimensional heat transfer model is defined as follows using the overall heat transfer coefficient. <Same as the present guidance></p> <p>(4) The overall heat transfer coefficient is obtained by a combination of heat transfer coefficient of convection, conduction and radiation, and the above case is defined as follows.</p> <p>(A) Overall heat transfer coefficient for Q_1 <Same as the present guidance></p> <p>(B) Overall heat transfer coefficient for Q_2 <Same as the present guidance></p> <p>(C) Heat transfer coefficient(h_C) of convection is calculated as follows. <Same as the present guidance></p> <p>(D) Heat transfer coefficient(h_R) of radiation is calculated as follows. <Same as the present guidance></p> <p>(5) The temperature of the compartment and members is obtained in the same way as the method for obtaining the steel temperature in the example of Figure 2.4 .</p> <p>(A) The heat flux through the outer hull is expressed as follows. <Same as the present guidance></p> <p>(B) In equilibrium, the steel temperature is calculated as follows. <Same as the present guidance></p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p>202. 2-Dimensional heat transfer analysis model</p> <p>1. In the analytical heat transfer analysis method, the heat transfer of the hull structure is considered in two dimensions. Figure 2.5 shows the heat transfer path at the sideshell and bottom of the hull. The shape and heat transfer direction of the adjacent fluid(atmosphere or seawater) shown in Figure 2.5 should be considered when performing analytical heat transfer analysis. <Omitted></p> <p>2. The equilibrium equation of the analytical two-dimensional heat transfer analysis is described as follows. <Omitted></p> <p>203. Basic heat transfer model</p> <p>1. The one- and two-dimensional heat transfer analysis models are described as conduction heat transfer, convection heat transfer and radiation heat transfer.</p> <p>2. Heat Transfer of Conduction</p> <p>(1) The heat transfer rate by conduction per unit area is described by the Fourier equation as follows. <Omitted></p> <p>(2) The heat flux for a real structure is described as follows. <Omitted></p> <p>3. Heat Transfer of Convection</p> <p>(1) Heat transfer coefficient of convection is described as follows. <Omitted></p> <p>(2) Natural convection uses the Nusselt number to calculate the convective heat transfer coefficient. <Omitted></p> <p>(A) In hull structure, natural convection occurs in open spaces and also occurs in enclosed cofferdams.</p> <p>(B) The Nusselt number for the vertical plate is obtained as shown in Table 2.1. <Omitted></p> <p>(C) Rayleigh number(R_a) is calculated as follows. <Omitted></p> <p>(D) Prandtl number(P_r) is calculated as follows. <Omitted></p> <p>(E) Grashof number(G_r) is calculated as follows. <Omitted></p> <p>(F) Nusselt number for horizontal plate is calculated as shown in Table 2.2 and Table 2.3. <Omitted></p>	<p>2. 2-Dimensional heat transfer analysis model</p> <p>(1) In the analytical heat transfer analysis method, the heat transfer of the hull structure is considered in two dimensions. Figure 2.5 shows the heat transfer path at the sideshell and bottom of the hull. The shape and heat transfer direction of the adjacent fluid(atmosphere or seawater) shown in Figure 2.5 should be considered when performing analytical heat transfer analysis. <Same as the present guidance></p> <p>(2) The equilibrium equation of the analytical two-dimensional heat transfer analysis is described as follows. <Same as the present guidance></p> <p>3. Basic heat transfer model</p> <p>(1) The one- and two-dimensional heat transfer analysis models are described as conduction heat transfer, convection heat transfer and radiation heat transfer.</p> <p>(2) Heat Transfer of Conduction</p> <p>(A) The heat transfer rate by conduction per unit area is described by the Fourier equation as follows. <Same as the present guidance></p> <p>(B) The heat flux for a real structure is described as follows. <Same as the present guidance></p> <p>(3) Heat Transfer of Convection</p> <p>(A) Heat transfer coefficient of convection is described as follows. <Same as the present guidance></p> <p>(B) Natural convection uses the Nusselt number to calculate the convective heat transfer coefficient. <Same as the present guidance></p> <p>(a) In hull structure, natural convection occurs in open spaces and also occurs in enclosed cofferdams.</p> <p>(b) The Nusselt number for the vertical plate is obtained as shown in Table 2.1. <Same as the present guidance></p> <p>(c) Rayleigh number(R_a) is calculated as follows. <Same as the present guidance></p> <p>(d) Prandtl number(P_r) is calculated as follows. <Same as the present guidance></p> <p>(e) Grashof number(G_r) is calculated as follows. <Same as the present guidance></p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p>(G) The Nusselt number for inclined plate in case of $\theta < 60^\circ$ is calculated by changing g of vertical Rayleigh number to $g \cos \theta$. <Omitted></p> <p>(3) Forced convection is calculated using the following formula (McAdams's formula) with the Nusselt number. <Omitted></p> <p>(4) Fin effect of stiffener</p> <p>(A) ships carrying liquefied gases in bulk/ships using liquefied gases as fuels includes longitudinal and transverse stiffeners. These stiffeners affect the convective heat transfer coefficient, and the relationship can be expressed as: <Omitted></p> <p>(B) The stiffeners act like fin and the fin effect can be expressed as: <Omitted></p> <p>(C) T-bars and angles should be replaced with flat bars considering only the web to consider the fin effect. <Omitted></p> <p>4. Radiation heat transfer</p> <p>(1) When heat transfer is performed by radiation, the relationship is as follows. <Omitted></p> <p style="text-align: center;">Section-3 Material Properties</p> <p>301. General</p> <p>1. The designer is responsible for getting the material properties used in the heat transfer analysis.</p> <p>2. The designer should evaluate the material properties including the cryogenic environment of liquefied gas.</p> <p>3. Material properties can be obtained through the material supplier, the promulgated experimental data or the material experiments. If this is difficult, the specified values from 302. to 305. can be used.</p>	<p>(g) The Nusselt number for inclined plate in case of $\theta < 60^\circ$ is calculated by changing g of vertical Rayleigh number to $g \cos \theta$. <Same as the present guidance></p> <p>(C) Forced convection is calculated using the following formula (McAdams's formula) with the Nusselt number. <Same as the present guidance></p> <p>(D) Fin effect of stiffener</p> <p>(a) ships carrying liquefied gases in bulk/ships using liquefied gases as fuels includes longitudinal and transverse stiffeners. These stiffeners affect the convective heat transfer coefficient, and the relationship can be expressed as: <Same as the present guidance></p> <p>(b) The stiffeners act like fin and the fin effect can be expressed as: <Same as the present guidance></p> <p>(c) T-bars and angles should be replaced with flat bars considering only the web to consider the fin effect. <Same as the present guidance></p> <p>(4) Radiation heat transfer</p> <p>(A) When heat transfer is performed by radiation, the relationship is as follows. <Same as the present guidance></p> <p>103. Material Properties</p> <p>1. General</p> <p>(1) The designer is responsible for getting the material properties used in the heat transfer analysis.</p> <p>(2) The designer should evaluate the material properties including the cryogenic environment of liquefied gas.</p> <p>(3) Material properties can be obtained through the material supplier, the promulgated experimental data or the material experiments. If this is difficult, the specified values from 2. to 5. can be used.</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p>302. Properties of steel</p> <p>1. Refer Table 2.4 for thermal conductivity of steel plate. <Omitted></p> <p>2. Refer Table 2.5 for emissivity of steel plate. <Omitted></p> <p>3. The thermal conductivity for stainless steel and Invar(36% Ni steel) is obtained from the following formula, and the values in Table 2.6 are used for the relevant constant. <Omitted></p> <p>303. Properties of seawater</p> <p>1. Refer Table 2.7 for density of seawater. <Omitted></p> <p>2. Refer Table 2.8 for specific heat of seawater. <Omitted></p> <p>3. Refer Table 2.9 for thermal conductivity of seawater. <Omitted></p> <p>4. Refer Table 2.10 for kinematic viscosity of seawater. <Omitted></p> <p>5. Refer Table 2.11 for prandtl number of seawater. <Omitted></p> <p>304. Properties of air</p> <p>1. Refer Table 2.12 for properties of air. <Omitted></p> <p>305. Properties of fresh water</p> <p>1. Refer Table 2.13 for properties of fresh water. <Omitted></p>	<p>2. Properties of steel</p> <p>(1) Refer Table 2.4 for thermal conductivity of steel plate. <Same as the present guidance></p> <p>(2) Refer Table 2.5 for emissivity of steel plate. <Same as the present guidance></p> <p>(3) The thermal conductivity for stainless steel and Invar(36% Ni steel) is obtained from the following formula, and the values in Table 2.6 are used for the relevant constant. <Same as the present guidance></p> <p>3. Properties of seawater</p> <p>(1) Refer Table 2.7 for density of seawater. <Same as the present guidance></p> <p>(2) Refer Table 2.8 for specific heat of seawater. <Same as the present guidance></p> <p>(3) Refer Table 2.9 for thermal conductivity of seawater. <Same as the present guidance></p> <p>(4) Refer Table 2.10 for kinematic viscosity of seawater. <Same as the present guidance></p> <p>(5) Refer Table 2.11 for prandtl number of seawater. <Same as the present guidance></p> <p>4. Properties of air</p> <p>(1) Refer Table 2.12 for properties of air. <Same as the present guidance></p> <p>5. Properties of fresh water</p> <p>(1) Refer Table 2.13 for properties of fresh water. <Same as the present guidance></p>	<p>To reflect request for establishment/revision of classification of technical rules</p>

Present	Amendment	Reason
<p style="text-align: center;">Section 4 Calculation Conditions</p> <p>40†. Calculation conditions</p> <p>†. To determine the grade of plate and sections used in the hull structure, a temperature calculation shall be performed for all tank types when the cargo temperature is below -10°C. The following assumptions shall be made in this calculation:</p> <p>(1) The loading condition of the ship for the calculation is to be full loaded condition.</p> <p>(2) Temperature distribution and heat transfer are to be dealt with as the phenomena in a steady state. No transient condition may be considered.</p> <p>(3) the primary barrier of all tanks shall be assumed to be at the cargo temperature;</p> <p>(4) The liquid cargo is to be assumed to have uniform temperature distribution.</p> <p>(5) In addition to (3), where a complete or partial secondary barrier is required, it shall be assumed to be at the cargo temperature at atmospheric pressure for any one tank only;</p> <p>(6) For worldwide service, ambient temperatures shall be taken as 5°C for air and 0°C for seawater. Higher values may be accepted for ships operating in restricted areas and, conversely, lower values may be fixed by the Society for ships trading to areas where lower temperatures are expected during the winter months. If necessary, refer to Table 2.14. <Omitted></p> <p>(7) still air and seawater conditions shall be assumed, I.e. no adjustment for forced convection;</p> <p>(8) Sea water is to be assumed to have a density of 1,025kg/m³ and a coagulation point of -2.5°C with physical properties compatible with those of fresh water for other items.</p> <p>(9) degradation of the thermal insulation properties over the life of the ship due to factors such as thermal and mechanical ageing, compaction, ship motion and tank vibrations shall be assumed;</p>	<p>104. Calculation Conditions</p> <p>1. Calculation conditions</p> <p>(1) To determine the grade of plate and sections used in the hull structure, a temperature calculation shall be performed for all tank types when the cargo temperature is below -10°C. The following assumptions shall be made in this calculation:</p> <p>(A) The loading condition of the ship for the calculation is to be full loaded condition.</p> <p>(B) Temperature distribution and heat transfer are to be dealt with as the phenomena in a steady state. No transient condition may be considered.</p> <p>(C) the primary barrier of all tanks shall be assumed to be at the cargo temperature;</p> <p>(D) The liquid cargo is to be assumed to have uniform temperature distribution.</p> <p>(E) In addition to (C), where a complete or partial secondary barrier is required, it shall be assumed to be at the cargo temperature at atmospheric pressure for any one tank only;</p> <p>(F) For worldwide service, ambient temperatures shall be taken as 5°C for air and 0°C for seawater. Higher values may be accepted for ships operating in restricted areas and, conversely, lower values may be fixed by the Society for ships trading to areas where lower temperatures are expected during the winter months. If necessary, refer to Table 2.14. <Same as the present guidance></p> <p>(G) still air and seawater conditions shall be assumed, I.e. no adjustment for forced convection;</p> <p>(H) Sea water is to be assumed to have a density of 1,025kg/m³ and a coagulation point of -2.5°C with physical properties compatible with those of fresh water for other items.</p> <p>(I) degradation of the thermal insulation properties over the life of the ship due to factors such as thermal and mechanical ageing, compaction, ship motion and tank vibrations shall be assumed;</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

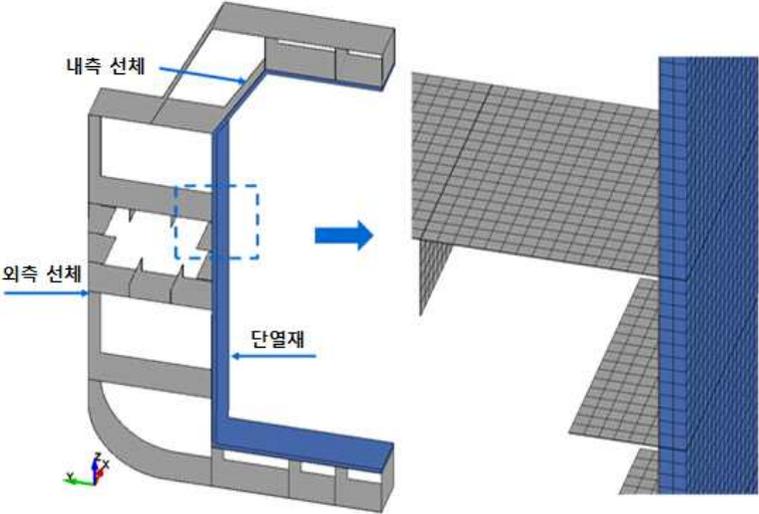
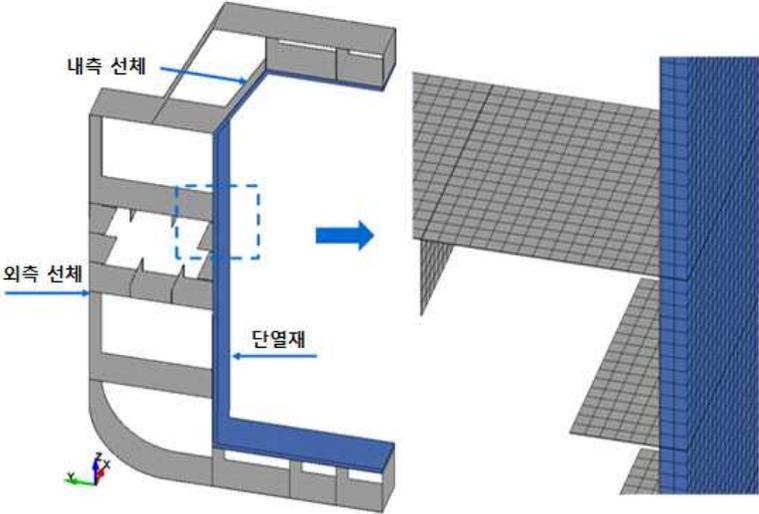
Present	Amendment	Reason
<p>(10) the cooling effect of the rising boil-off vapour from the leaked cargo shall be taken into account, where applicable;</p> <p>(11) credit for hull heating may be taken in accordance with 402.1, provided the heating arrangements are in compliance with 402.2;</p> <p>(12) no credit shall be given for any means of heating, except as described in 402.1;</p> <p>(13) The structures in hold space such as insulation materials and supports are to be assumed that they do not absorb liquid cargo.</p> <p>(14) In compartments where gases exist other than in hold spaces, it is to be assumed that they are in natural convection.</p> <p>(15) It is to be assumed that the gas and liquid within the same compartment are at the same temperature.</p> <p>(16) It is to be assumed that there is no transfer of gases within the insulation materials.</p> <p>(17) It is to be assumed that there is no influence of moisture.</p> <p>(18) It is to be assumed that there is no influence of paints.</p> <p>(19) The overall heat transfer coefficients at various boundaries can be used with the numeral values given in Table 2.15 of the Guidances, but calculation may be carried out by using empirical equations given in the heat transfer engineering data which has been made public. In this case, heat transfer due to radiation is also to be taken into account. <Omitted></p> <p>(20) The substance for which temperature distribution is investigated to be assumed to be of homogeneous one without directivity.</p> <p>(21) Frames may be dealt with as fins.</p> <p>(22) In case where hold spaces located forward and afterward the hold space under study are in the same locations, they may be treated as a two dimensional problem.</p>	<p>(J) the cooling effect of the rising boil-off vapour from the leaked cargo shall be taken into account, where applicable;</p> <p>(K) credit for hull heating may be taken in accordance with 2.1, provided the heating arrangements are in compliance with 2.2;</p> <p>(L) no credit shall be given for any means of heating, except as described in 2.1;</p> <p>(M) The structures in hold space such as insulation materials and supports are to be assumed that they do not absorb liquid cargo.</p> <p>(N) In compartments where gases exist other than in hold spaces, it is to be assumed that they are in natural convection.</p> <p>(O) It is to be assumed that the gas and liquid within the same compartment are at the same temperature.</p> <p>(P) It is to be assumed that there is no transfer of gases within the insulation materials.</p> <p>(Q) It is to be assumed that there is no influence of moisture.</p> <p>(R) It is to be assumed that there is no influence of paints.</p> <p>(S) The overall heat transfer coefficients at various boundaries can be used with the numeral values given in Table 2.15 of the Guidances, but calculation may be carried out by using empirical equations given in the heat transfer engineering data which has been made public. In this case, heat transfer due to radiation is also to be taken into account. <Same as the present guidance></p> <p>(T) The substance for which temperature distribution is investigated to be assumed to be of homogeneous one without directivity.</p> <p>(U) Frames may be dealt with as fins.</p> <p>(V) In case where hold spaces located forward and afterward the hold space under study are in the same locations, they may be treated as a two dimensional problem.</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

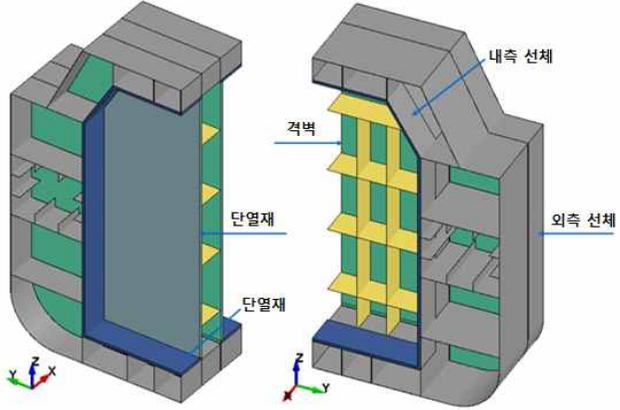
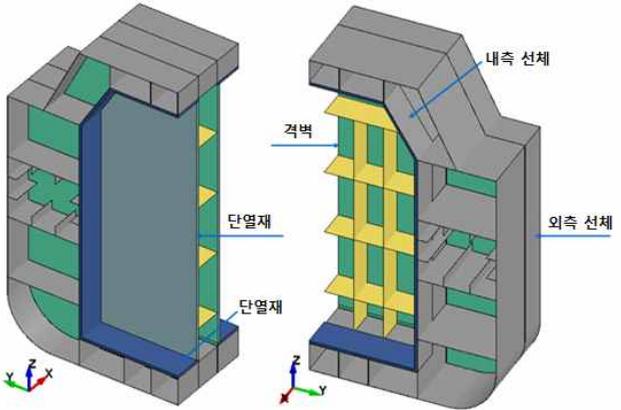
Present	Amendment	Reason
<p>2. At the upright cargo leakage is to be considered for the calculation in accordance with the following (1) to (5). However, no leakage may be considered for integral tanks and type C independent tanks.</p> <p>(1) It is to be assumed that the failure of all cargo tanks located between transverse watertight bulkheads are caused. However, in case where the cross section of the ship is divided into more than one compartments by longitudinal bulkheads of the ship, it is to be assumed that the failure of all cargo tanks within each such compartment is caused.</p> <p>(2) It is to be assumed that the locations of the failure of the cargo tank cover all conceivable ones.</p> <p>(3) It is to be assumed that only the liquid cargo leaks out where the cargo tank, supports and hull remain intact without involving any deflections or fracture.</p> <p>(4) For cargo tanks where the complete secondary barrier is required, it is to be assumed that the leakage of liquid cargo occurs instantaneously and the levels of residual liquid cargo in damaged cargo tank and the leaked liquid level in the hold space reach the same level instantaneously.</p> <p>(5) The temperature of the secondary barrier in a state of leakage is to be assumed to be the same as the cargo temperature at the atmospheric pressure, whereas the temperature of the intact cargo tank is the design temperature. The ship is to be assumed to stay upright.</p> <p>3. Secondary barrier shall also meet functional requirements at static heel condition of 30°.</p>	<p><u>(2)</u> At the upright cargo leakage is to be considered for the calculation in accordance with the following <u>(A)</u> to <u>(E)</u>. However, no leakage may be considered for integral tanks and type C independent tanks.</p> <p><u>(A)</u> It is to be assumed that the failure of all cargo tanks located between transverse watertight bulkheads are caused. However, in case where the cross section of the ship is divided into more than one compartments by longitudinal bulkheads of the ship, it is to be assumed that the failure of all cargo tanks within each such compartment is caused.</p> <p><u>(B)</u> It is to be assumed that the locations of the failure of the cargo tank cover all conceivable ones.</p> <p><u>(C)</u> It is to be assumed that only the liquid cargo leaks out where the cargo tank, supports and hull remain intact without involving any deflections or fracture.</p> <p><u>(D)</u> For cargo tanks where the complete secondary barrier is required, it is to be assumed that the leakage of liquid cargo occurs instantaneously and the levels of residual liquid cargo in damaged cargo tank and the leaked liquid level in the hold space reach the same level instantaneously.</p> <p><u>(E)</u> The temperature of the secondary barrier in a state of leakage is to be assumed to be the same as the cargo temperature at the atmospheric pressure, whereas the temperature of the intact cargo tank is the design temperature. The ship is to be assumed to stay upright.</p> <p><u>(3)</u> Secondary barrier shall also meet functional requirements at static heel condition of 30°.</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

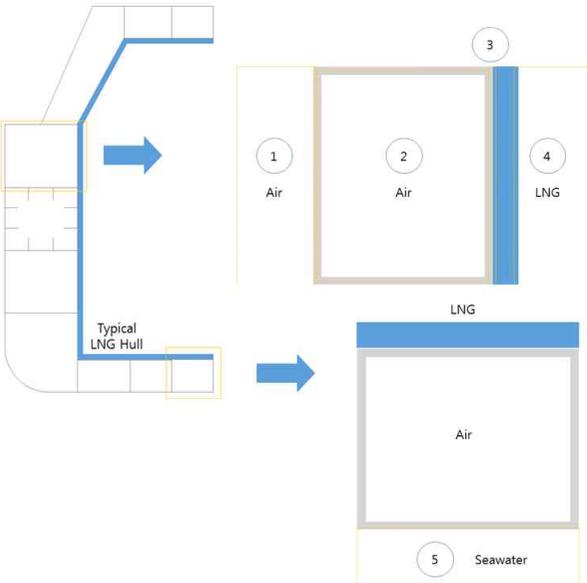
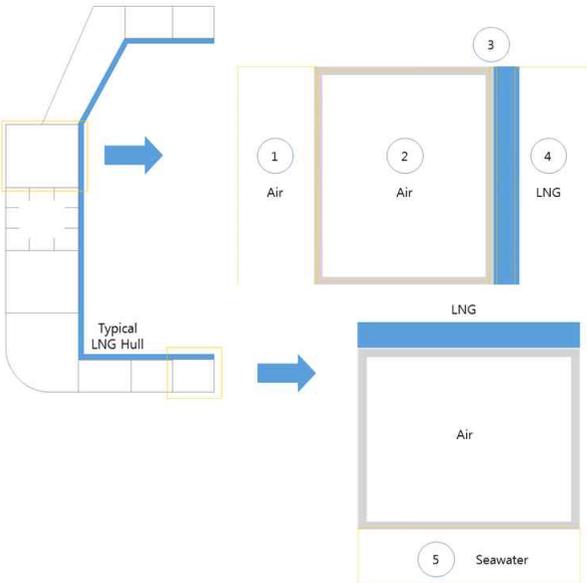
Present	Amendment	Reason
<p>402. Heating device</p> <p>1. Means of heating structural materials may be used to ensure that the material temperature does not fall below the minimum allowed for the grade of material specified in Table 2.16. In the calculations required in 401.1, credit for such heating may be taken in accordance with the following:</p> <p>(1) for any transverse hull structure;</p> <p>(2) for longitudinal hull structure referred to in 401.2 where colder ambient temperatures are specified, provided the material remains suitable for the ambient temperature conditions of 5°C for air and 0°C for seawater with no credit taken in the calculations for heating; and</p> <p>(3) as an alternative to (2), for longitudinal bulkhead between cargo tanks, credit may be taken for heating, provided the material remain suitable for a minimum design temperature of -30°C, or a temperature 30°C lower than that determined by 401.1 with the heating considered, whichever is less.</p> <p>2. The means of heating referred to in 1 shall comply with the following requirements:</p> <p>(1) the heating system shall be arranged so that, in the event of failure in any part of the system, standby heating can be maintained equal to not less than 100% of the theoretical heat requirement;</p> <p>(2) the heating system shall be considered as an essential auxiliary. All electrical components of at least one of the systems provided in accordance with 1(1) shall be supplied from the emergency source of electrical power; and</p> <p>(3) the design and construction of the heating system shall be included in the approval of the containment system by the Society.</p>	<p>2. Heating device</p> <p>(1) Means of heating structural materials may be used to ensure that the material temperature does not fall below the minimum allowed for the grade of material specified in Table 2.16. In the calculations required in 1. (1), credit for such heating may be taken in accordance with the following:</p> <p>(A) for any transverse hull structure;</p> <p>(B) for longitudinal hull structure referred to in 1. (2) where colder ambient temperatures are specified, provided the material remains suitable for the ambient temperature conditions of 5°C for air and 0°C for seawater with no credit taken in the calculations for heating; and</p> <p>(C) as an alternative to (B), for longitudinal bulkhead between cargo tanks, credit may be taken for heating, provided the material remain suitable for a minimum design temperature of -30°C, or a temperature 30°C lower than that determined by 1. (1) with the heating considered, whichever is less.</p> <p>(2) The means of heating referred to in (1) shall comply with the following requirements:</p> <p>(A) the heating system shall be arranged so that, in the event of failure in any part of the system, standby heating can be maintained equal to not less than 100% of the theoretical heat requirement;</p> <p>(B) the heating system shall be considered as an essential auxiliary. All electrical components of at least one of the systems provided in accordance with (1)(A) shall be supplied from the emergency source of electrical power; and</p> <p>(C) the design and construction of the heating system shall be included in the approval of the containment system by the Society.</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

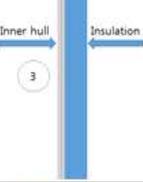
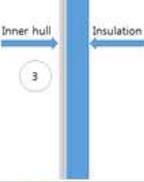
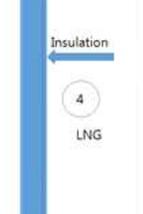
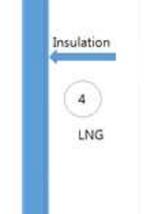
Present	Amendment	Reason
<p style="text-align: center;">Section 5 Result Derivation</p> <p>501. General</p> <p>1. The steel grade of the structural members connecting the inner hull to the outer hull are determined using the average temperature.</p> <p>2. The temperature of structural members is to be represented by the temperature at their half thickness, and for individual members, the following requirements (1) through (4) are to be complied with :</p> <p>(1) The temperature of those frames fitted to plates is to be assumed to be the same as the temperature of the plates, but when the temperature distribution of the frame in the direction of depth is known, the area mean of the temperature distribution may be taken.</p> <p>(2) The temperature of web frames supporting frames or plates is to be the temperature at their half depth for webs, and the temperature of face plates for these.</p> <p>(3) The temperature of members connecting the inner shall and outer shell, e.g., brackets and girders is to be of the mean of the temperature of the inner shell and that of the outer shell.</p> <p>(4) The temperature of brackets is to be the temperature at their centroid.</p> <p>502. Selection of steel grade</p> <p>1. The grade of plate and sections used in the hull structure shall be selected in accordance with a temperature calculation when the cargo temperature is below -10°C.</p> <p>2. The shell and deck plating of the ship and all stiffeners attached thereto shall be in accordance with the requirements of Pt 3 of the Rules, if the calculated temperature of the material in the design condition is below -5°C due to the influence of the cargo temperature, the material shall be in accordance with Table 2.16.</p>	<p>105. Result Derivation</p> <p>1. General</p> <p><u>(1)</u> The steel grade of the structural members connecting the inner hull to the outer hull are determined using the average temperature.</p> <p><u>(2)</u> The temperature of structural members is to be represented by the temperature at their half thickness, and for individual members, the following requirements <u>(A)</u> through <u>(D)</u> are to be complied with :</p> <p><u>(A)</u> The temperature of those frames fitted to plates is to be assumed to be the same as the temperature of the plates, but when the temperature distribution of the frame in the direction of depth is known, the area mean of the temperature distribution may be taken.</p> <p><u>(B)</u> The temperature of web frames supporting frames or plates is to be the temperature at their half depth for webs, and the temperature of face plates for these.</p> <p><u>(C)</u> The temperature of members connecting the inner shall and outer shell, e.g., brackets and girders is to be of the mean of the temperature of the inner shell and that of the outer shell.</p> <p><u>(D)</u> The temperature of brackets is to be the temperature at their centroid.</p> <p>2. Selection of steel grade</p> <p><u>(1)</u> The grade of plate and sections used in the hull structure shall be selected in accordance with a temperature calculation when the cargo temperature is below -10°C.</p> <p><u>(2)</u> The shell and deck plating of the ship and all stiffeners attached thereto shall be in accordance with the requirements of Pt 3 of the Rules, if the calculated temperature of the material in the design condition is below -5°C due to the influence of the cargo temperature, the material shall be in accordance with Table 2.16.</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

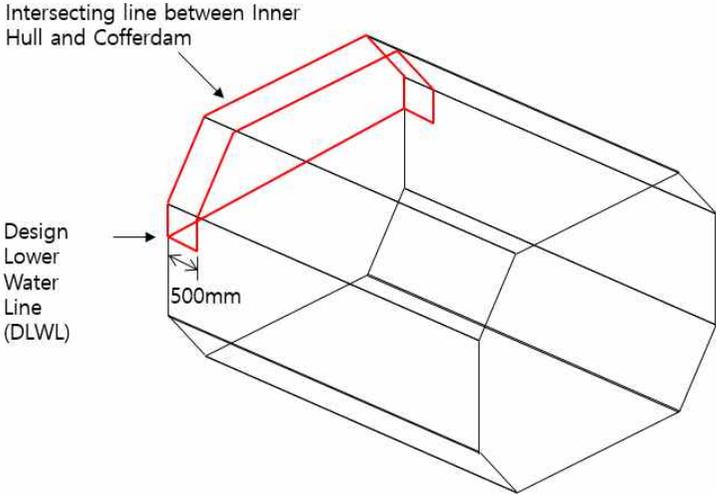
Present	Amendment	Reason
<p>3. The materials of all other hull structures for which the calculated temperature in the design condition is below 0°C, due to the influence of cargo temperature and that do not form the secondary barrier, shall also be in accordance with Table 2.16. This includes hull structure supporting the cargo tanks, inner bottom plating, longitudinal bulkhead plating, transverse bulkhead plating, floors, webs, stringers and all attached stiffening members. <Omitted></p> <p>4. According to USCG code, the deck stringer and sheer strake must be at least Grade E steel. The strake at the turn of the bilge must be Grade D or Grade E. Application range is to follow Table 2.17. <Omitted></p> <p>503. Selection of welding consumables</p> <p>1. Application of welding consumables for welded joints of various grades of steel is to be as specified in Table 2.18. <Omitted></p> <p>2. Welding consumables for lower toughness of steel may be used for welded joints of different toughness of steel of the same specified strength.</p> <p>3. In case of welding of steels of different specified strength, the welding consumables required for the steel of lower specified strength may be used, provided that adequate means for preventing cracks are considered.</p> <p>4. It is recommended that controlled low hydrogen type consumables are to be used when joining higher strength structural steel to the same or lower strength level, except that other consumables may be used at the discretion of the Society when the carbon equivalent is below or equal to 0.41%. When other than controlled low hydrogen type electrodes are used, appropriate procedure tests for hydrogen cracking may be conducted at the discretion of the Society.</p>	<p><u>(3)</u> The materials of all other hull structures for which the calculated temperature in the design condition is below 0°C, due to the influence of cargo temperature and that do not form the secondary barrier, shall also be in accordance with Table 2.16. This includes hull structure supporting the cargo tanks, inner bottom plating, longitudinal bulkhead plating, transverse bulkhead plating, floors, webs, stringers and all attached stiffening members. <Same as the present guidance></p> <p><u>(4)</u> According to USCG code, the deck stringer and sheer strake must be at least Grade E steel. The strake at the turn of the bilge must be Grade D or Grade E. Application range is to follow Table 2.17. <Same as the present guidance></p> <p>3. Selection of welding consumables</p> <p><u>(1)</u> Application of welding consumables for welded joints of various grades of steel is to be as specified in Table 2.18. <Same as the present guidance></p> <p><u>(2)</u> Welding consumables for lower toughness of steel may be used for welded joints of different toughness of steel of the same specified strength.</p> <p><u>(3)</u> In case of welding of steels of different specified strength, the welding consumables required for the steel of lower specified strength may be used, provided that adequate means for preventing cracks are considered.</p> <p><u>(4)</u> It is recommended that controlled low hydrogen type consumables are to be used when joining higher strength structural steel to the same or lower strength level, except that other consumables may be used at the discretion of the Society when the carbon equivalent is below or equal to 0.41%. When other than controlled low hydrogen type electrodes are used, appropriate procedure tests for hydrogen cracking may be conducted at the discretion of the Society.</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p style="text-align: center;">CHAPTER 3 FEM HEAT TRANSFER ANALYSIS</p> <p style="text-align: center;">Section 1 Modeling</p> <p>101. 2-Dimensional heat transfer</p> <p>1. The two-dimensional heat transfer analysis model is performed using a solid element or a shell element. When it is necessary to consider the temperature distribution in the thickness direction, the solid element should be used.</p> <p>2. When solid elements are used, the mesh size shall not be greater than 200mm*200mm and shall be divided into two or more elements in the thickness direction. An example of a two-dimensional heat transfer model is shown in Figure 3-1. <Omitted></p> <p>3. When using a shell element, the mesh size should be 200mm*200mm or less.</p>  <p style="text-align: center;">Figure 3.1 Model for 2-dimensional heat transfer analysis</p>	<p style="text-align: center;">Section 2 FEM HEAT TRANSFER ANALYSIS</p> <p>201. Modeling</p> <p>1. 2-Dimensional heat transfer</p> <p>(1) The two-dimensional heat transfer analysis model is performed using a solid element or a shell element. When it is necessary to consider the temperature distribution in the thickness direction, the solid element should be used.</p> <p>(2) When solid elements are used, the mesh size shall not be greater than 200mm*200mm and shall be divided into two or more elements in the thickness direction. An example of a two-dimensional heat transfer model is shown in Figure 2.8. <Same as the present guidance></p> <p>(3) When using a shell element, the mesh size should be 200mm*200mm or less.</p>  <p style="text-align: center;">Figure 2.8 Model for 2-dimensional heat transfer analysis</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p>102. 3-Dimensional heat transfer</p> <ol style="list-style-type: none"> 1. When the analysis considering cofferdams is required, the heat transfer analysis model should be extended to both sides of the bulkhead of the cofferdam, and three-dimensional heat transfer analysis considering the length direction of the hull should be performed. 2. The insulation and bulkhead are modeled on the LNG side of the inner hull. 3. The transverse bulkhead should be considered in the finite element heat transfer analysis model. The bulkhead should only be insulated on the LNG contact surface and have a stiffener on the other side. 4. The three-dimensional heat transfer analysis model is performed using a solid element or a shell element. When it is necessary to consider the temperature distribution in the thickness direction, the solid element should be used. 5. When solid element are used, the mesh size shall not be greater than 200mm*200mm and shall be divided into two or more elements in the thickness direction. An example of a three-dimensional heat transfer model is shown in Figure 3.2. <Omitted>  <p>Figure 3.2 Model for 3-dimensional heat transfer analysis</p>	<p>2. 3-Dimensional heat transfer</p> <ol style="list-style-type: none"> (1) When the analysis considering cofferdams is required, the heat transfer analysis model should be extended to both sides of the bulkhead of the cofferdam, and three-dimensional heat transfer analysis considering the length direction of the hull should be performed. (2) The insulation and bulkhead are modeled on the LNG side of the inner hull. (3) The transverse bulkhead should be considered in the finite element heat transfer analysis model. The bulkhead should only be insulated on the LNG contact surface and have a stiffener on the other side. (4) The three-dimensional heat transfer analysis model is performed using a solid element or a shell element. When it is necessary to consider the temperature distribution in the thickness direction, the solid element should be used. (5) When solid element are used, the mesh size shall not be greater than 200mm*200mm and shall be divided into two or more elements in the thickness direction. An example of a three-dimensional heat transfer model is shown in Figure 2.9. <Same as the present guidance>  <p>Figure 2.9 Model for 3-dimensional heat transfer analysis</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p>6. When using a shell element, the mesh size should be 200mm*200mm or less.</p> <p style="text-align: center;">Section 2 Material Properties</p> <p>201. General</p> <p>1. Follow Chapter 2, Section 3.</p> <p style="text-align: center;">Section 3 Calculation Conditions</p> <p>301. General</p> <p>1. Follow Chapter 2, Section 4.</p> <p>2. Convection, radiation and conduction according to the environment of each member should be considered as shown in Figure 3-3 and Table 3-1. <Omitted></p> <p>3. The temperature and heat transfer coefficient in Table 3-1 shall be entered base on the results of Chapter 3.</p>  <p style="text-align: center;">Figure 3-3 Finite element modeling in heat transfer analysis of liquefied gas carrier hull</p>	<p>(6) When using a shell element, the mesh size should be 200mm*200mm or less.</p> <p><u>202. Material Properties</u></p> <p>1. General</p> <p>(1) Follow Chapter 2, Section 1 103..</p> <p><u>203. Calculation Conditions</u></p> <p>1. General</p> <p>(1) Follow Chapter 2, Section 1 104..</p> <p>(2) Convection, radiation and conduction according to the environment of each member should be considered as shown in Figure 2.10 and Table 2.19. <Same as the present guidance></p> <p>(3) The temperature and heat transfer coefficient in Table 2.19 shall be entered base on the results of Chapter 2, Section 2.</p>  <p style="text-align: center;">Figure 2.10 Finite element modeling in heat transfer analysis of liquefied gas carrier hull</p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present			Amendment			Reason
Table 3.1 Heat transfer process in heat transfer analysis			Table 2.19 Heat transfer process in heat transfer analysis			
Structural part	Heat transfer process	Input for FEM analysis	Structural part	Heat transfer process	Input for FEM analysis	To reflect request for establishment/revision of classification technical rules
	Radiation	Air temperature Emissivity of outer hull surface		Radiation	Air temperature Emissivity of outer hull surface	
	Convection	Air temperature Convective heat transfer coefficient		Convection	Air temperature Convective heat transfer coefficient	
	Conduction	Not considered		Conduction	Not considered	
	Radiation	Air temperature of compartment View factor of compartment surface Emissivity of compartment surface		Radiation	Air temperature of compartment View factor of compartment surface Emissivity of compartment surface	
	Convection	Air temperature of compartment Convective heat transfer coefficient		Convection	Air temperature of compartment Convective heat transfer coefficient	
	Conduction	Thermal conductivity and specific heat of steel		Conduction	Thermal conductivity and specific heat of steel	
	Radiation	Not considered		Radiation	Not considered	
	Convection	Not considered		Convection	Not considered	
	Conduction	Thermal conductivity and specific heat of steel and insulation		Conduction	Thermal conductivity and specific heat of steel and insulation	
	Radiation	Not considered The temperature of liquefied gas is applied on the secondary barrier.		Radiation	Not considered The temperature of liquefied gas is applied on the secondary barrier.	
	Convection	Not considered The temperature of liquefied gas is applied on the secondary barrier.		Convection	Not considered The temperature of liquefied gas is applied on the secondary barrier.	
	Conduction	Not considered The temperature of liquefied gas is applied on the secondary barrier.		Conduction	Not considered The temperature of liquefied gas is applied on the secondary barrier.	
	Radiation	Not considered		Radiation	Not considered	
	Convection	Seawater temperature Convective heat transfer coefficient		Convection	Seawater temperature Convective heat transfer coefficient	
	Conduction	Not considered		Conduction	Not considered	

Present	Amendment	Reason
<p style="text-align: center;">Section 4 Result Derivation</p> <p>401. General † Follow Chapter 2 Section 5-50†.</p> <p>402. Selection of steel grade † Follow Chapter 2 Section 5-502.</p> <p>403. Selection of welding consumable</p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 5-503. 2. Selection of welding consumable is based on the steel grade determined using the average temperature of the member. 	<p>204. Result Derivation</p> <ol style="list-style-type: none"> 1. General <ol style="list-style-type: none"> (1) Follow Chapter 2 Section 1 105. 1. 2. Selection of steel grade <ol style="list-style-type: none"> (1) Follow Chapter 2 Section 1 105. 2. (2) <u>As shown in Figure 2.11, the steel of cofferdam surrounded by the design lower water line above and intersecting line between inner hull and cofferdam and steel inside 500mm from intersecting line between inner hull and cofferdam should be selected based on the temperature of mid-section of membrane tank.</u> <div style="text-align: center;">  </div> <p style="text-align: center;">Figure 2.11 Important consideration range in steel selection</p> <ol style="list-style-type: none"> 3. Selection of welding consumable <ol style="list-style-type: none"> (1) Follow Chapter 2 Section 1 105. 3. (2) Selection of welding consumable is based on the steel grade determined using the average temperature of the member. 	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p><New></p>	<p style="text-align: center;">CHAPTER 3 HEAT TRANSFER ANALYSIS FOR INDEPENDENT TYPE A TANK</p> <p style="text-align: center;"><u>Section 1 Analytical Heat Transfer Analysis</u></p> <p>101. Analysis Procedure <u>1. Follow Chapter 2 Section 1 101..</u></p> <p>102. Modeling <u>1. Follow Chapter 2 Section 1 102..</u></p> <p>103. Material Properties <u>1. Follow Chapter 2 Section 1 103..</u></p> <p>104. Calculation Conditions <u>1. Follow Chapter 2 Section 1 104..</u></p> <p>105. Result Derivation <u>1. Follow Chapter 2 Section 1 105..</u> <u>2. Figure 3.1 illustrates calculation results performed for a midship section of a type A LNG Carrier using analytical method.</u></p>	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason								
<p>⟨New⟩</p>	<div data-bbox="958 225 1464 783" data-label="Figure"> <p>IMO Condition:</p> <table border="1"> <tr> <td>Airtemp.</td> <td>5 °C</td> </tr> <tr> <td>Sea water temp.</td> <td>0 °C</td> </tr> <tr> <td>Wind Speed</td> <td>0 knots</td> </tr> <tr> <td>LNG temp.</td> <td>-163 °C</td> </tr> </table> </div> <p data-bbox="817 810 1662 839">Figure 3.1 Temperature calculation for Type A tank using analytical method</p>	Airtemp.	5 °C	Sea water temp.	0 °C	Wind Speed	0 knots	LNG temp.	-163 °C	<p data-bbox="1852 268 2116 464">To reflect request for establishment/revision of classification technical rules</p>
Airtemp.	5 °C									
Sea water temp.	0 °C									
Wind Speed	0 knots									
LNG temp.	-163 °C									

Present	Amendment	Reason
<p>〈New〉</p>	<p style="text-align: center;"><u>Section 2 FEM HEAT TRANSFER ANALYSIS</u></p> <p><u>201. Modeling</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 201.. 2. <u>The heat transfer of conduction through the supports(including vertical, anti rolling, anti pitching and anti floating) connecting the cargo tank with the inner hull should be considered.</u> <p><u>202. Material Properties</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 202.. <p><u>203. Calculation Conditions</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 203.. <p><u>204. Result Derivation</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 204.. 2. <u>A sample 2D model of the Type A hull for heat transfer analysis is presented in Figure 3.2.</u> 3. <u>As shown 'd' in Figure 3.3., the steel on the secondary barrier should be extended 500mm toward the centerline from the intersection between the deck plate and a line at a static heel of ±30 degrees, inside the top side tank and hopper tank.</u> 	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
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<New>

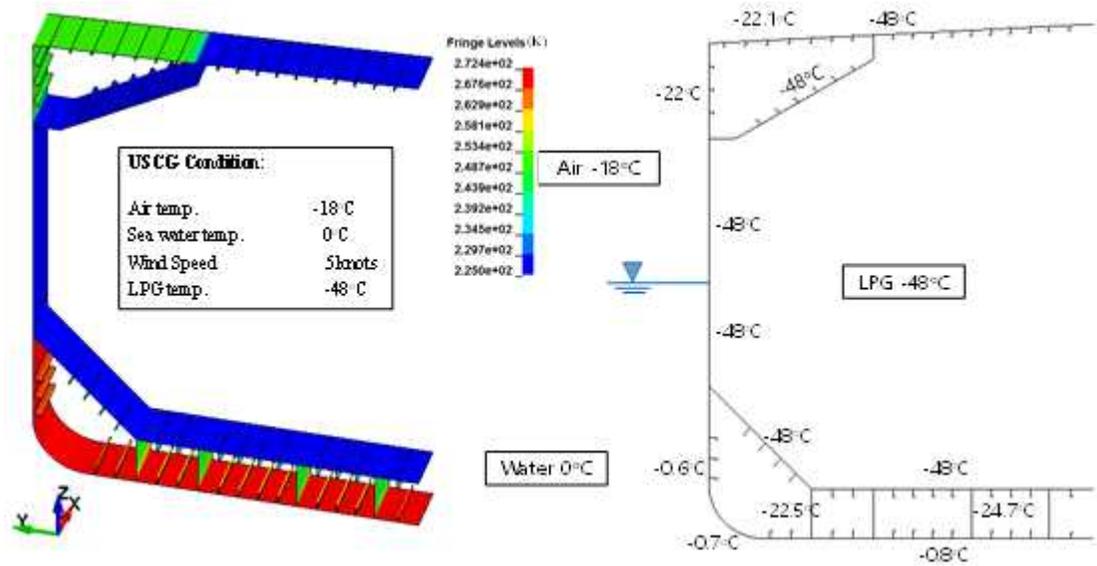


Figure 3.2 Temperature calculation for Type A tank using 2D FEM

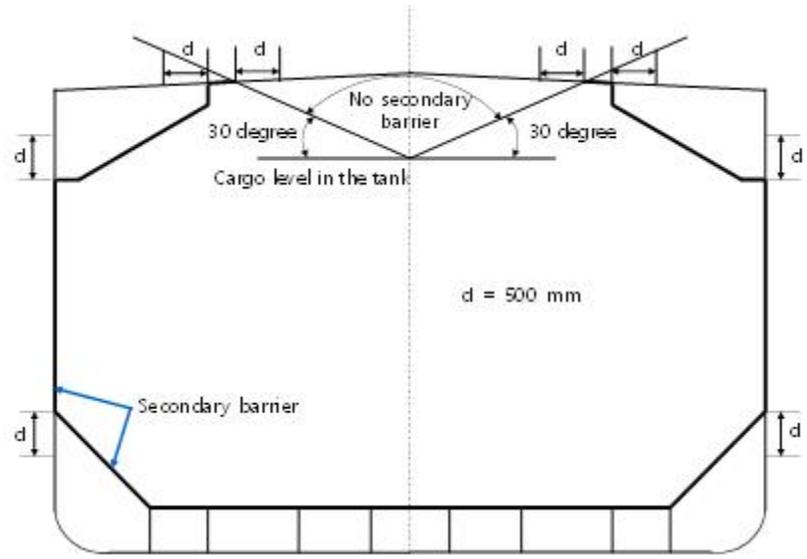


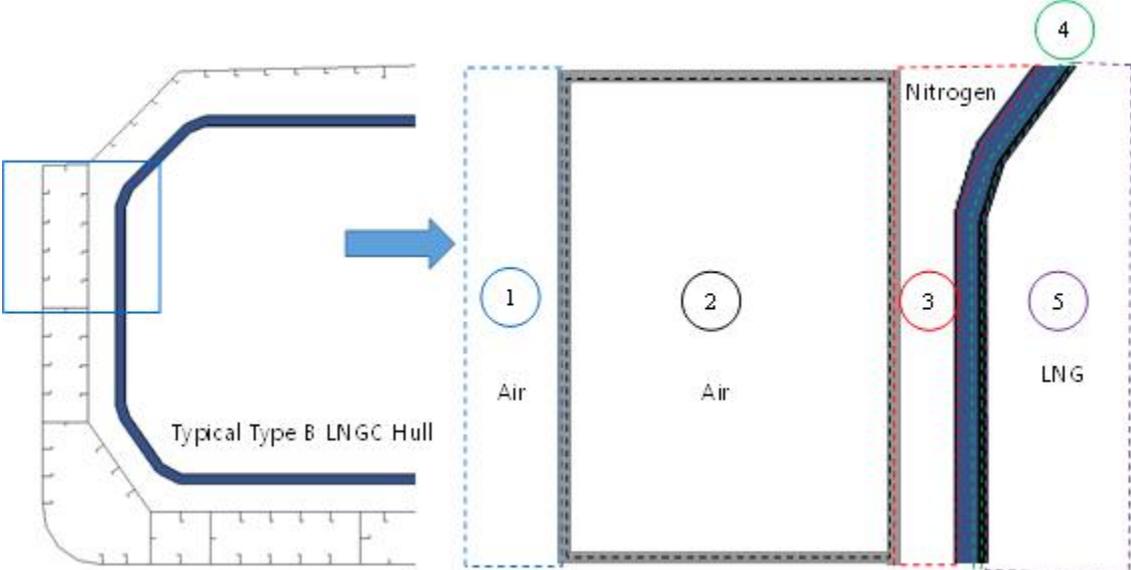
Figure 3.3 Application range extension of secondary steel for Type A tank

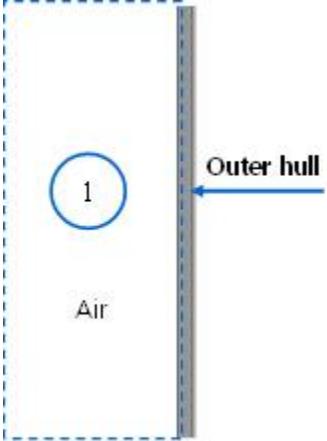
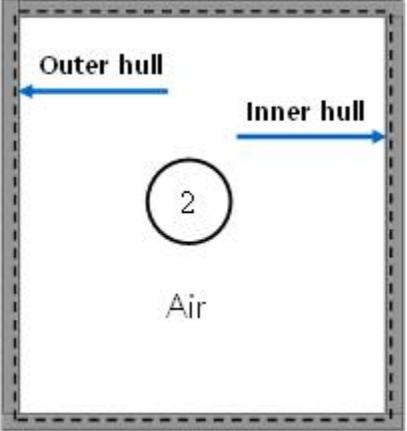
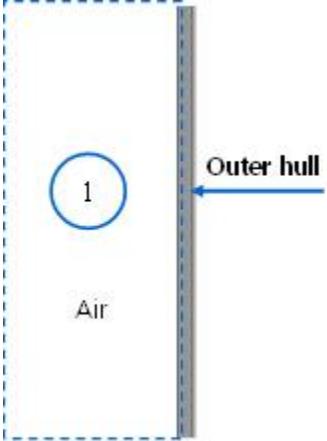
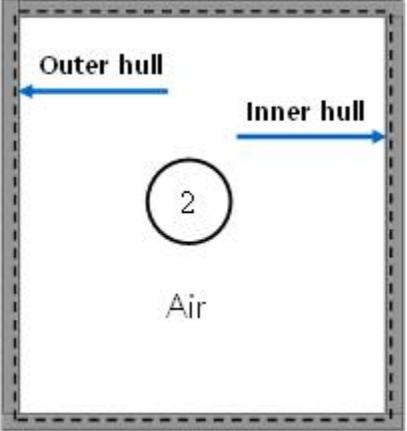
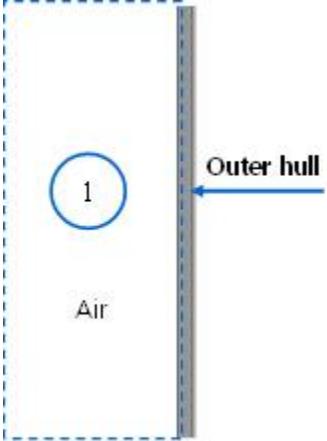
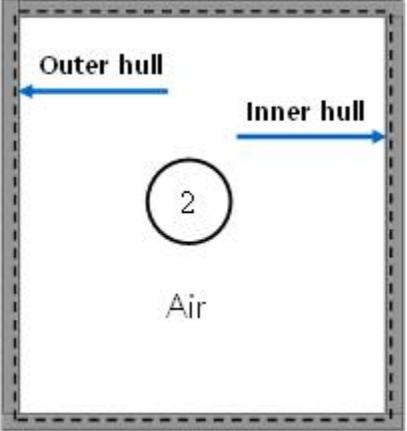
To reflect request for establishment/revision of classification technical rules

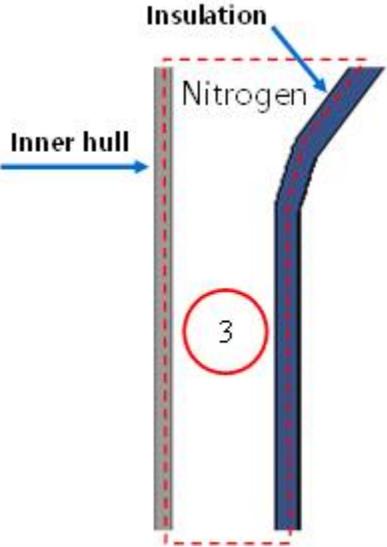
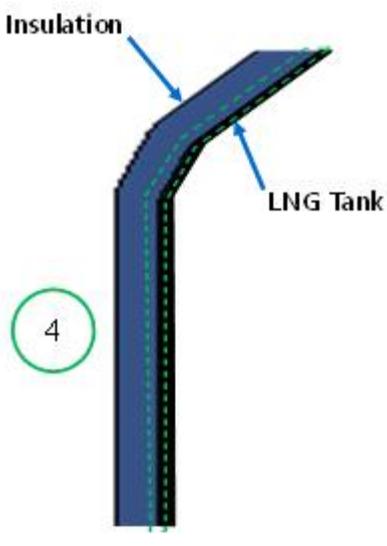
Present	Amendment	Reason
<p>⟨New⟩</p>	<p align="center">CHAPTER 4 HEAT TRANSFER ANALYSIS FOR INDEPENDENT TYPE B TANK</p> <p align="center"><u>Section 1 Analytical Heat Transfer Analysis</u></p> <p><u>101. Analysis Procedure</u> 1. <u>Follow Chapter 2 Section 1 101..</u></p> <p><u>102. Modeling</u> 1. <u>Follow Chapter 2 Section 1 102..</u></p> <p align="center">-</p> <p><u>103. Material Properties</u> 1. <u>Follow Chapter 2 Section 1 103..</u></p> <p><u>104. Calculation Conditions</u> 1. <u>Follow Chapter 2 Section 1 104..</u></p> <p><u>105. Result Derivation</u> 1. <u>Follow Chapter 2 Section 1 105..</u> 2. <u>Figure 4.1 illustrates a temperature calculation results performed for a midship section of a Type B LNG carrier using analytical method.</u></p>	<p>To reflect request for establishment/revision of classification technical rules</p>

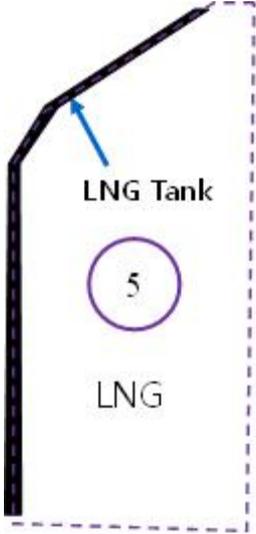
Present	Amendment	Reason
<p data-bbox="174 181 257 209">〈New〉</p>	<div data-bbox="958 229 1496 858" data-label="Figure"> <p data-bbox="1243 459 1489 609">USCG Condition: Air temp. -18°C Sea water temp. 0°C Wind Speed 5 knots LNG temp. -163°C</p> </div> <p data-bbox="817 887 1662 914">Figure 4.1 Temperature calculation for Type B tank using analytical method</p>	<p data-bbox="1850 225 2110 424">To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p>〈New〉</p>	<p style="text-align: center;"><u>Section 2 FEM HEAT TRANSFER ANALYSIS</u></p> <p><u>201. Modeling</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 201. 2. <u>The temperature calculation of independent type B tank is similar with the membrane type, but need to consider the gap between inner hull and the cargo tank. The heat transfer of conduction through the supports(including vertical, anti rolling, anti pitching and anti floating) connecting the cargo tank with the inner hull should be considered.</u> <p><u>202. Material Properties</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 202. <p><u>203. Calculation Conditions</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 203. 2. <u>Figure 4.2 and Table 4.1 presents the application of FEM to modelling of overall heat transfer in the independent type B tank and the required input data for each form of heat energy transfer.</u> <p><u>204. Result Derivation</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 204. 2. <u>Figure 4-3 is one example of temperature analysis result for 2D FEM midship section.</u> 3. <u>Figure 4-4 illustrates a temperature calculation results performed for 3D FEM including cofferdam.</u> 	<p>To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<p data-bbox="203 180 282 209">〈New〉</p>	 <p data-bbox="698 798 1839 863">Figure 4.2 Finite element modeling in heat transfer analysis of hull with independent type B tank at temperature calculation</p>	<p data-bbox="1877 268 2130 467">To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason																		
<New>	Table 4.1 Heat transfer process in heat transfer analysis for hull with independent type B tank	To reflect request for establishment/revision of classification technical rules																		
	<table border="1" style="width: 100%; border-collapse: collapse;"> <thead> <tr> <th style="width: 50%;">Parts</th> <th style="width: 25%;">Heat Transfer Process</th> <th style="width: 25%;">Required input data in FEM</th> </tr> </thead> <tbody> <tr> <td data-bbox="604 328 1207 804" rowspan="3">  </td> <td data-bbox="1207 328 1464 405">Radiation</td> <td data-bbox="1464 328 1870 405">Air temperature Outer hull surface emissivity</td> </tr> <tr> <td data-bbox="1207 405 1464 513">Convection</td> <td data-bbox="1464 405 1870 513">Air temperature Convective heat transfer coefficient</td> </tr> <tr> <td data-bbox="1207 513 1464 804">Conduction</td> <td data-bbox="1464 513 1870 804">Not considered</td> </tr> <tr> <td data-bbox="604 804 1207 1251" rowspan="3">  </td> <td data-bbox="1207 804 1464 912">Radiation</td> <td data-bbox="1464 804 1870 912">View factor of the enclosure surfaces Emissivity of enclosure surfaces</td> </tr> <tr> <td data-bbox="1207 912 1464 1059">Convection</td> <td data-bbox="1464 912 1870 1059">Air properties such as conductivity and specific heat Convection heat transfer coefficient</td> </tr> <tr> <td data-bbox="1207 1059 1464 1251">Conduction</td> <td data-bbox="1464 1059 1870 1251">Steel conductivity and specific heat</td> </tr> </tbody> </table>		Parts	Heat Transfer Process	Required input data in FEM		Radiation	Air temperature Outer hull surface emissivity	Convection	Air temperature Convective heat transfer coefficient	Conduction	Not considered		Radiation	View factor of the enclosure surfaces Emissivity of enclosure surfaces	Convection	Air properties such as conductivity and specific heat Convection heat transfer coefficient	Conduction	Steel conductivity and specific heat	
	Parts		Heat Transfer Process	Required input data in FEM																
			Radiation	Air temperature Outer hull surface emissivity																
			Convection	Air temperature Convective heat transfer coefficient																
			Conduction	Not considered																
	Radiation	View factor of the enclosure surfaces Emissivity of enclosure surfaces																		
	Convection	Air properties such as conductivity and specific heat Convection heat transfer coefficient																		
	Conduction	Steel conductivity and specific heat																		

Present	Amendment			Reason
<New>		Radiation	View factor of the enclosure surface Emissivity of enclosure surface	To reflect request for establishment/revision of classification technical rules
		Convection	Nitrogen properties such as conductivity and specific heat Convection heat transfer coefficient	
		Conduction	Not considered	
		Radiation	Not considered	
		Convection	Not considered	
		Conduction	Thermal material properties of insulation and LNG tank material such as conductivity and specific heat	

Present	Amendment		Reason	
<New>		Radiation	Not considered	To reflect request for establishment/revision of classification technical rules
		Convection	Not considered The temperature of liquefied gas is applied on the primary barrier.	
		Conduction	Not considered The temperature of liquefied gas is applied on the primary barrier.	

Present	Amendment	Reason
<p data-bbox="203 180 282 209">〈New〉</p>	<div data-bbox="705 199 1803 774"> </div> <p data-bbox="795 778 1713 805">Figure 4-3 Temperature distribution calculation by 2D FEM heat transfer analysis</p> <div data-bbox="705 853 1803 1316"> </div> <p data-bbox="705 1332 1803 1364">Figure 4-4 Temperature distribution in the cofferdam calculated by 3D FEM heat transfer analysis</p>	<p data-bbox="1877 263 2139 470">To reflect request for establishment/revision of classification technical rules</p>

Present	Amendment	Reason
<New>	<p style="text-align: center;">CHAPTER 5 HEAT TRANSFER ANALYSIS FOR INDEPENDENT TYPE C TANK</p> <p style="text-align: center;"><u>Section 1 Analytical Heat Transfer Analysis</u></p> <p><u>101. Analysis Procedure</u> 1. <u>Follow Chapter 2 Section 1 101..</u></p> <p><u>102. Modeling</u> 1. <u>Follow Chapter 2 Section 1 102..</u></p> <p><u>103. Material Properties</u> 1. <u>Follow Chapter 2 Section 1 103..</u></p> <p><u>104. Calculation Conditions</u> 1. <u>Follow Chapter 2 Section 1 104..</u></p> <p><u>105. Result Derivation</u> 1. <u>Follow Chapter 2 Section 1 105..</u></p>	To reflect request for establishment/revision of classification technical rules

Present	Amendment	Reason
<p>〈New〉</p>	<p style="text-align: center;"><u>Section 2 FEM HEAT TRANSFER ANALYSIS</u></p> <p><u>201. Modeling</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 201.. 2. <u>The heat transfer of conduction through the support(cradle support or sliding support) should be considered. In the case of bilobe, heat transfer of conduction through the cradle support and anti-floating support should be considered.</u> <p><u>202. Material Properties</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 202.. <p><u>203. Calculation Conditions</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 203.. <p><u>204. Result Derivation</u></p> <ol style="list-style-type: none"> 1. Follow Chapter 2 Section 2 204.. 	<p>To reflect request for establishment/revision of classification technical rules</p>